

# **Measurement and analysis of active mirror slope errors under operating conditions**

John P. Sutter, Simon G. Alcock & Kawal Sawhney

Diamond Light Source Ltd, Harwell Science and Innovation Campus,  
Chilton, Didcot, Oxfordshire OX11 0DE, United Kingdom



The quality of active optics may be measured

- ex-situ: in a special metrology laboratory
  - precise test of quality under carefully controlled conditions
- in-situ: under realistic operating conditions

Both types of measurements are performed on active mirrors at Diamond Light Source:

- ex-situ: in our in-house metrology laboratory equipped with
  - BESSY-type NOM
  - Fizeau interferometer
  - micro-roughness interferometer
- **in-situ: on the synchrotron beamlines where the mirrors are installed, using only equipment that is already installed or can be installed without breaking vacuum.**

Sometimes ex-situ and in-situ measurements support each other.

Sometimes in-situ measurements provide vital information that ex-situ measurements cannot.

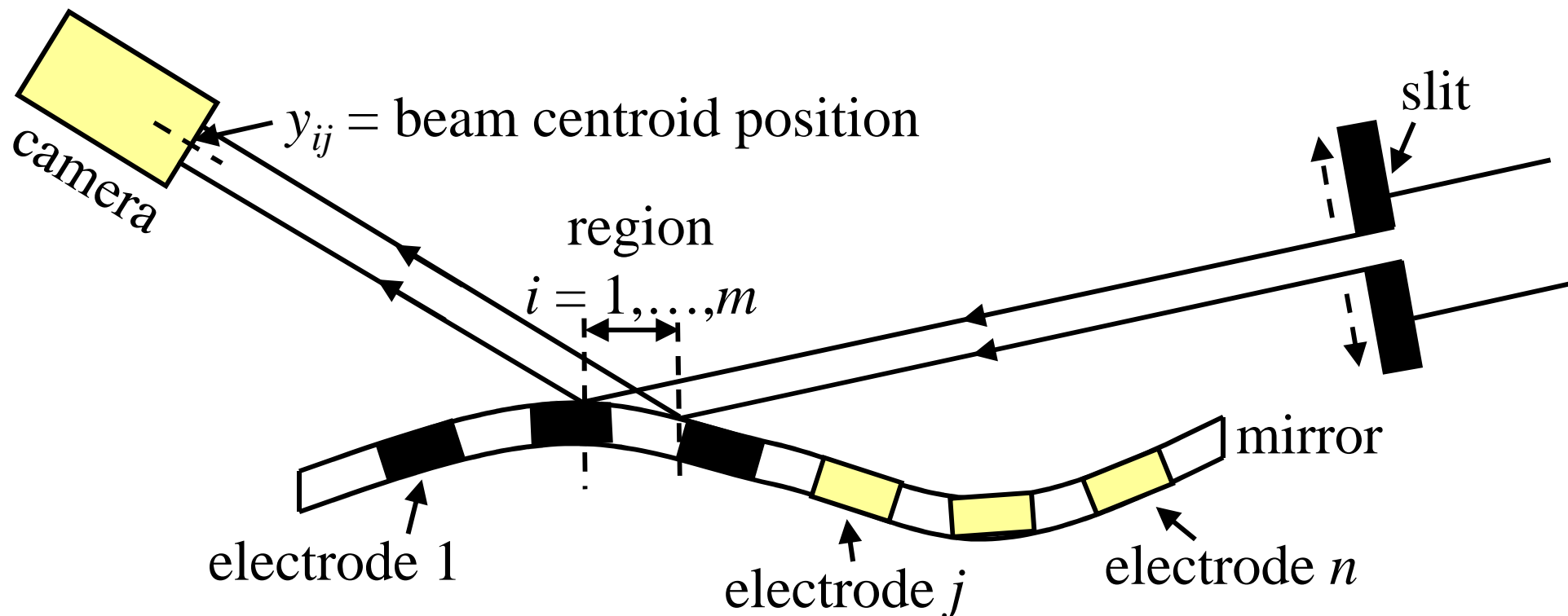
**Examples of both cases will be presented here.**

Only well-known methods have been used to collect the data:  
pencil-beam scans + interaction matrix calculations.

**But at Diamond Light Source,  
they have been applied to new problems:**

- surface stability of bimorph mirrors
- discovery and remediation of surface defects
- diagnosis of malfunctioning actuators
- optimisation of a *collimating* mirror.

## Pencil beam method as performed in-situ on bimorph mirrors:

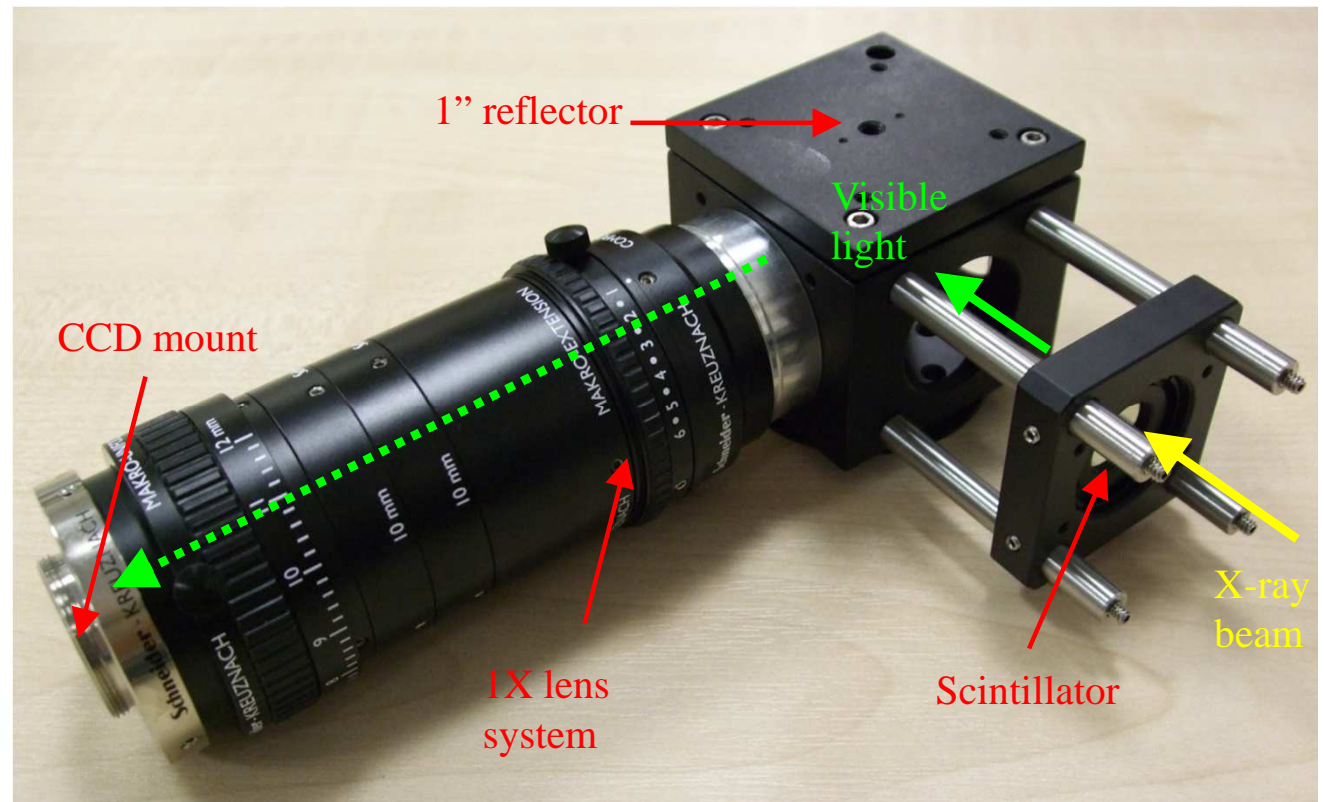
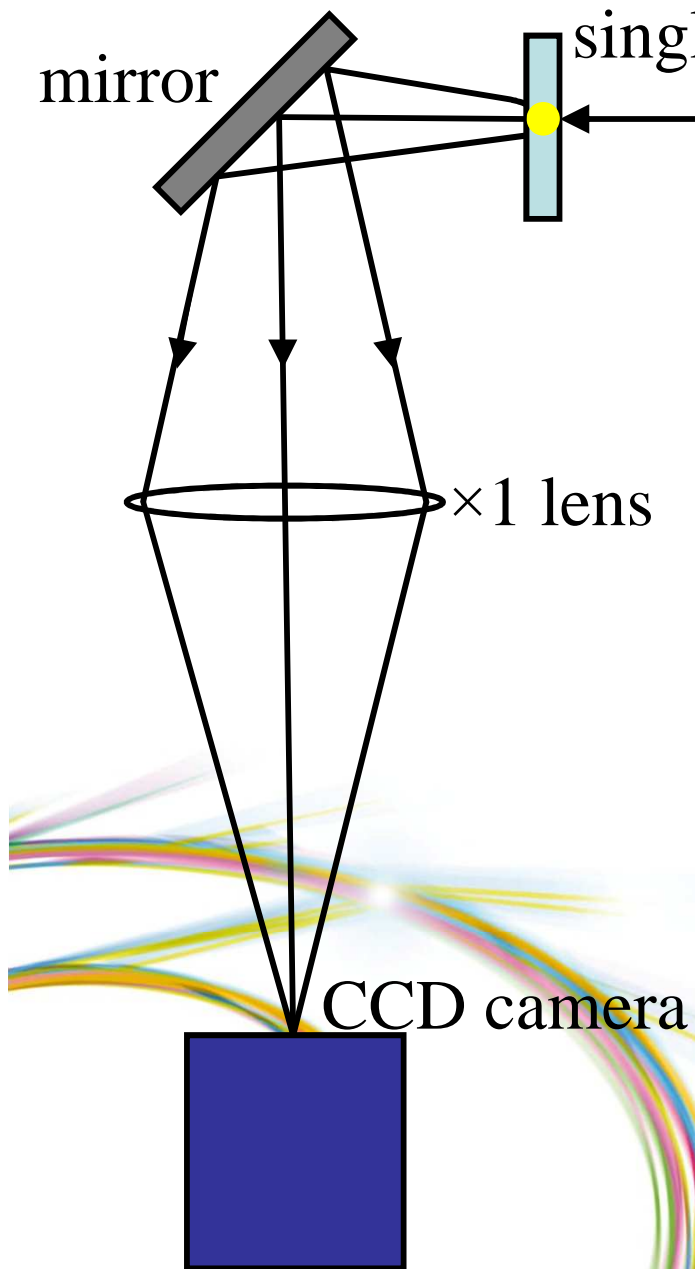


Scan a small ( $\sim 10 \mu\text{m}$ ) slit across incident beam.  
Record beam position in camera.

The “camera” may be a real imaging device or a detector behind a scannable slit.

# X-ray camera designed by our Diagnostics group:

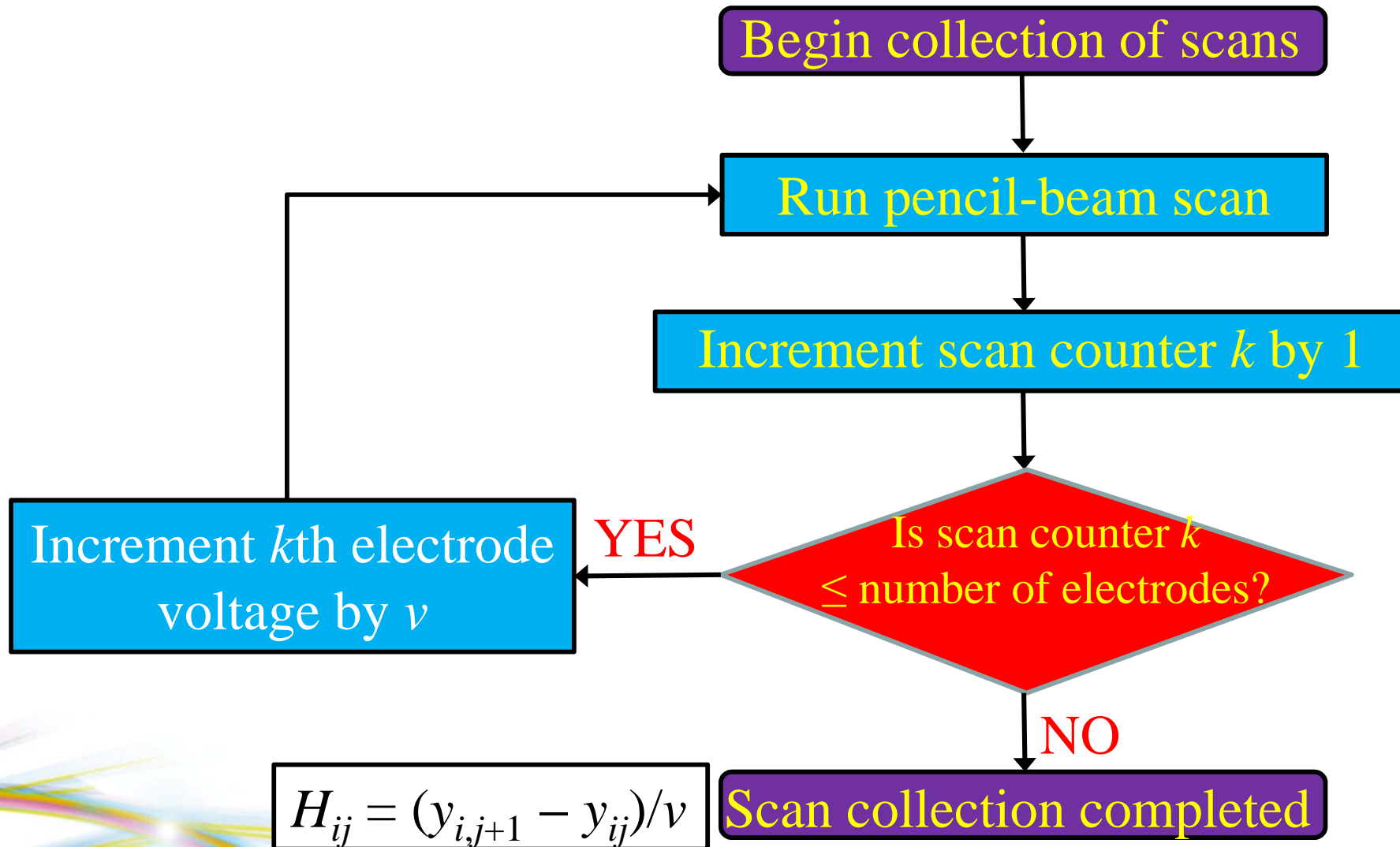
This camera is small and portable –  
used at several beamlines that lack their own imaging systems.



Camera image transmitted on Firewire/GigE.

90% integrated line spread  
function  $\sim 6.35 \mu\text{m}$

Interaction matrix  $H$ : Shows each actuator's effect on figure.



Vector  $Y$ :  $Y_i = i$ th correction to reflected beam position

Vector  $V$ :  $V_j =$  voltage correction for  $j$ th electrode

Shortest length least squares solution:  $V = H^\dagger Y$   
 $H^\dagger =$  Moore-Penrose pseudoinverse of  $H$

diamond



**An automated procedure now exists to perform the pencil beam scans and interaction matrix inversion.**

**Only standard software packages were used:**

Motors and X-ray camera image collection were controlled through EPICS.

Pencil-beam scans were executed and analyzed using the Generic Data Acquisition (GDA) package.

Jython scripts were used to

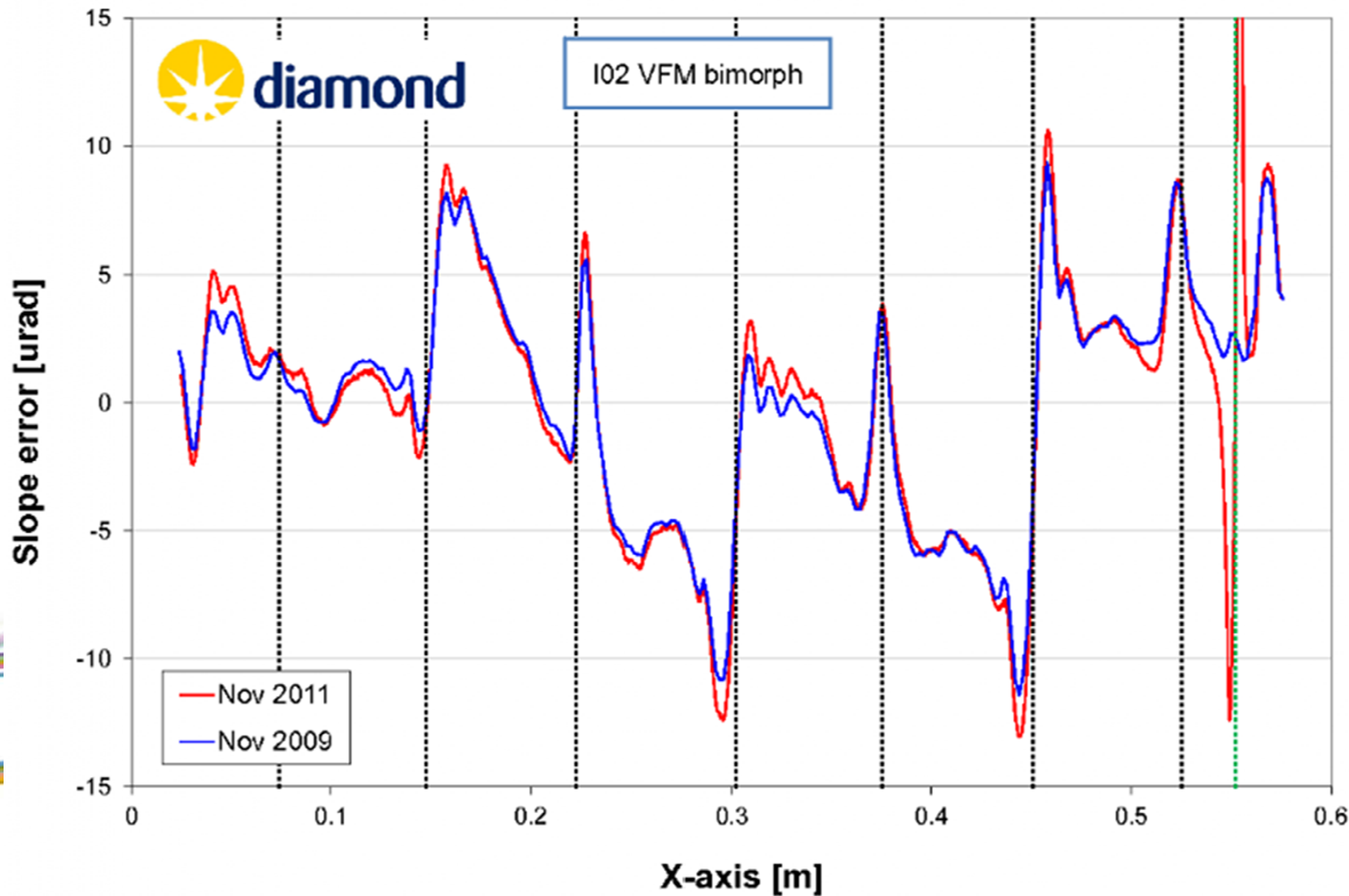
- calculate the beam centroid position using a 2-D Gaussian fit
- calculate and invert the interaction matrix.

NOTE: the centroid calculation is reproducible to within 0.1 pixel → the line-spread function does not limit the resolution.

In the following, the angular resolution  $\sim 0.1 \mu\text{rad}$ .

J. Sutter, S. Alcock & K. Sawhney, *Proc. SPIE* **8139**, 813906 (2011)

Ex-situ measurements revealed a “junction effect” in bimorph mirrors:  
violent jumps in slope error at the junctions of the piezoelectric plates:



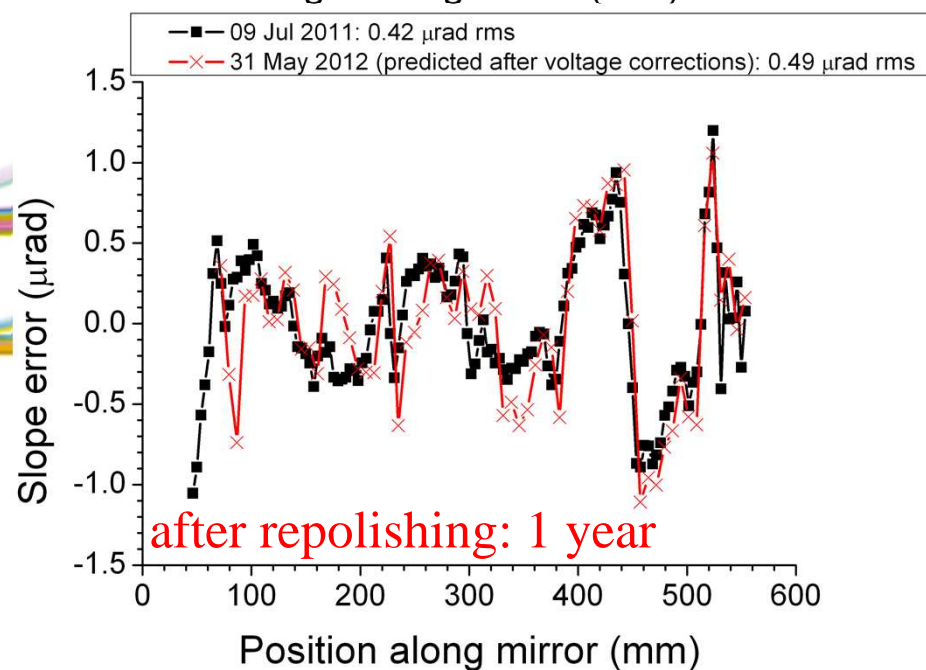
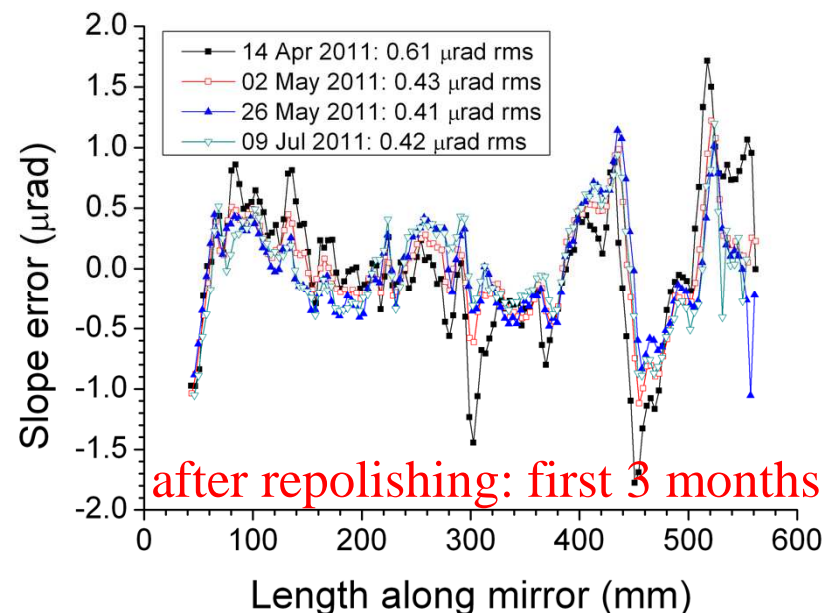
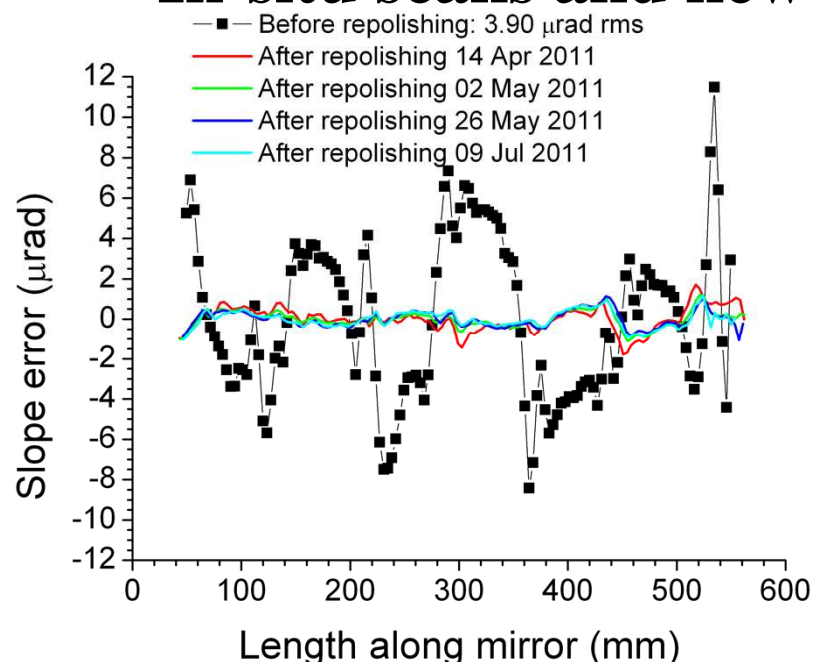
Measured on  
Diamond NOM

Dotted lines are junctions.

Figure stable over 2 years of operation!

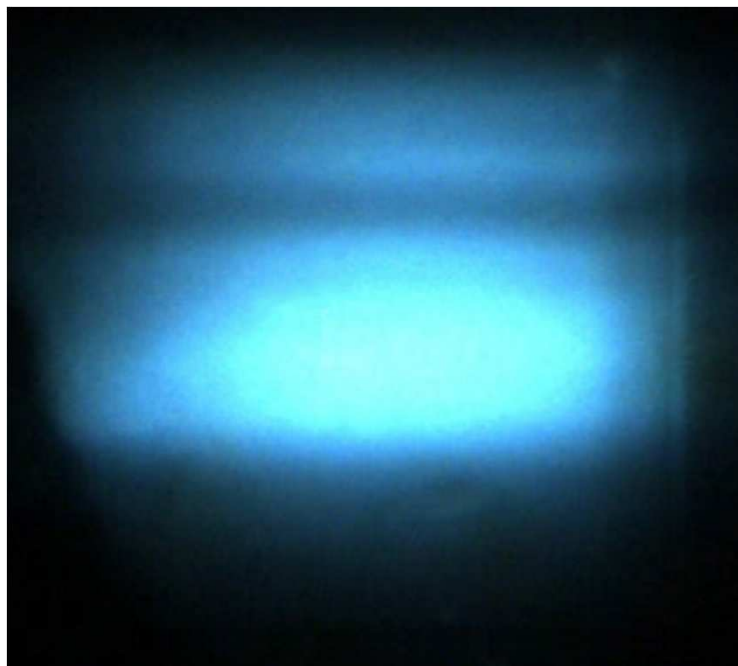


**In-situ measurements agree with these ex-situ results.**  
**Corrugation was removed on two sets of KB mirrors by repolishing.**  
**In-situ scans and new beam images confirmed success.**

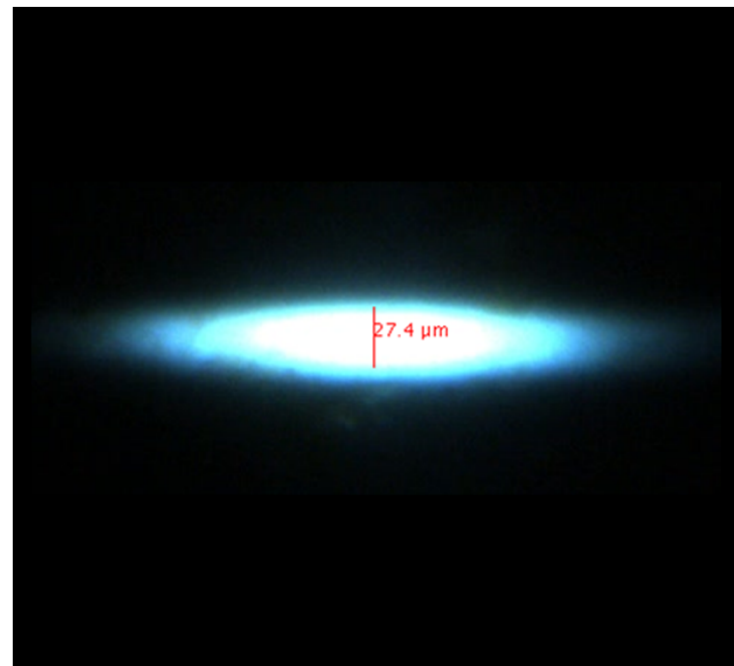


**Vertical focusing mirror  
at I04 before repolishing,  
at I03 after repolishing.**

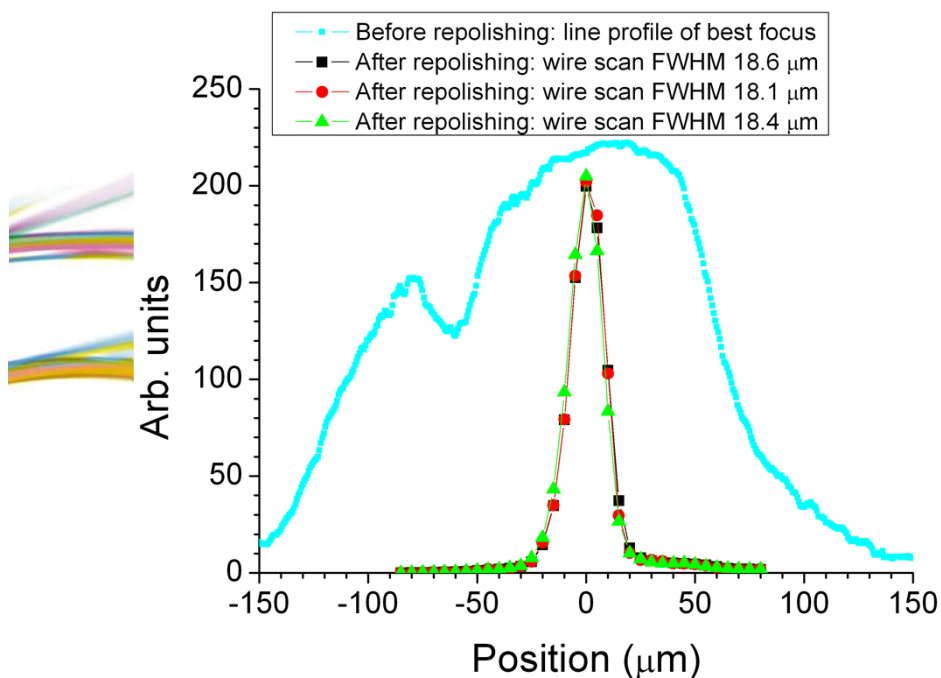
**Surface after repolishing was  
stable over 1 full year.**



Before at I04



Immediately after re-installation of VFM at I03



Horizontal focusing mirror  
was likewise re-polished  
and re-installed.

Horizontal beam width reduced  
from 120  $\mu\text{m}$  to 70  $\mu\text{m}$   
(theoretical 65  $\mu\text{m}$ )

# Pencil-beam scans also help optimise mechanically bent mirrors!

## Examples at Diamond Light Source:

- I15 (Extreme Conditions)
- I20 (X-ray Spectroscopy)

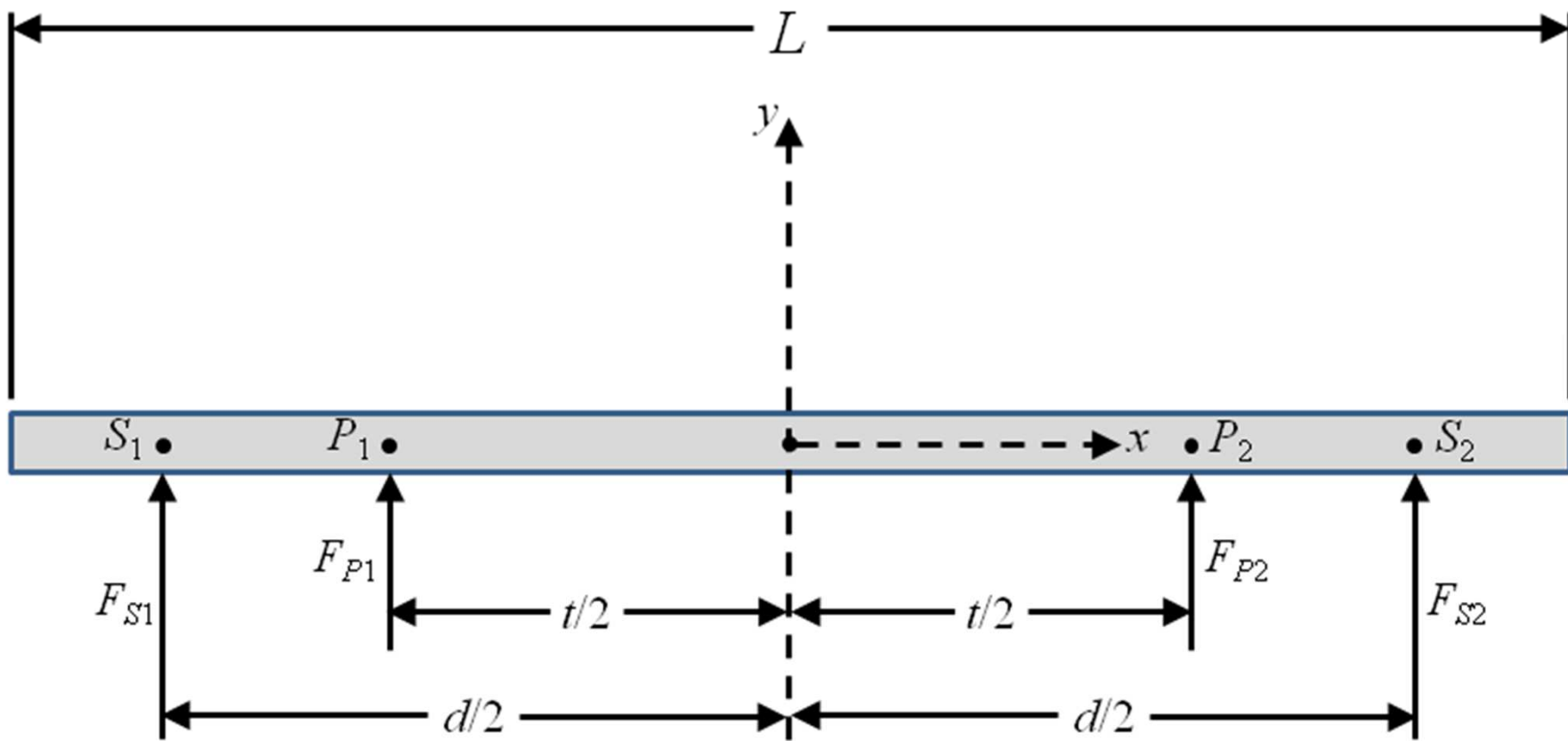
These mirrors are flat, rectangular slabs of silicon with two bending actuators, one at each end.

The bending actuators deform the mirror into the desired shape: elliptical for focusing (I15), parabolic for collimation (I20).

**But gravity adds an error-producing “sag” deformation!**

To compensate, an upward force is applied by an actuator to two symmetrically positioned “props”:





Standard vertically focusing mirror:  
Fixed support points  $S_1$  and  $S_2$ ; prop points  $P_1$  and  $P_2$

Sag compensation actuator is placed in a bulky frame of preloaded springs

→ ex-situ measurements require mirror to be removed from this frame.

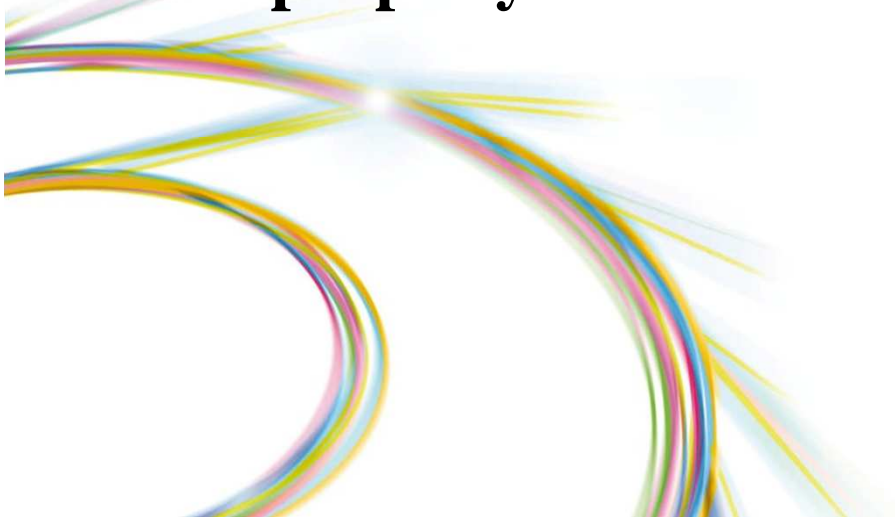
## **Ex-situ measurements are *not* helpful for finding the best setting of the sag compensation actuator!**

In theory, finite element analysis can simulate mirror deformation versus setting of sag compensation actuator.

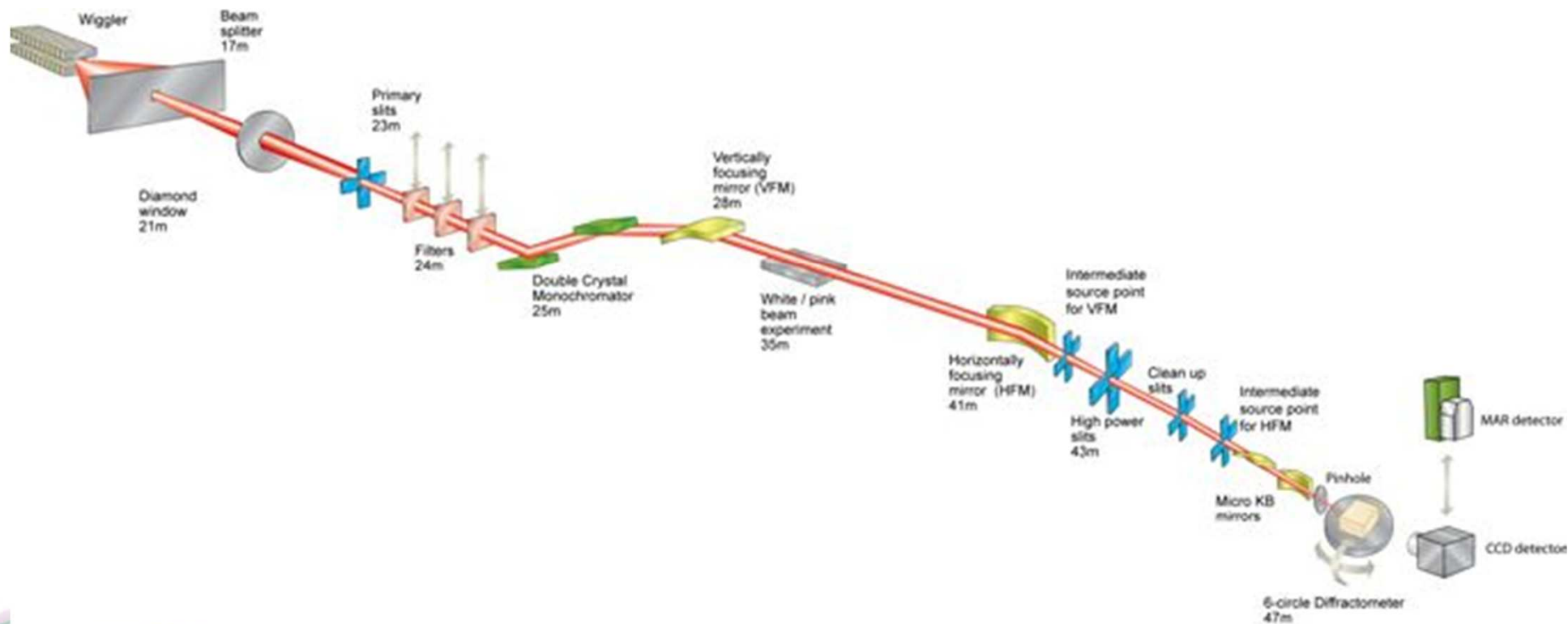
In practice,

- this calibration is often unknown or highly uncertain.
- even if measured in air, the calibration may be different in vacuum.

**In-situ measurements are the only way to know that the mirror works properly.**

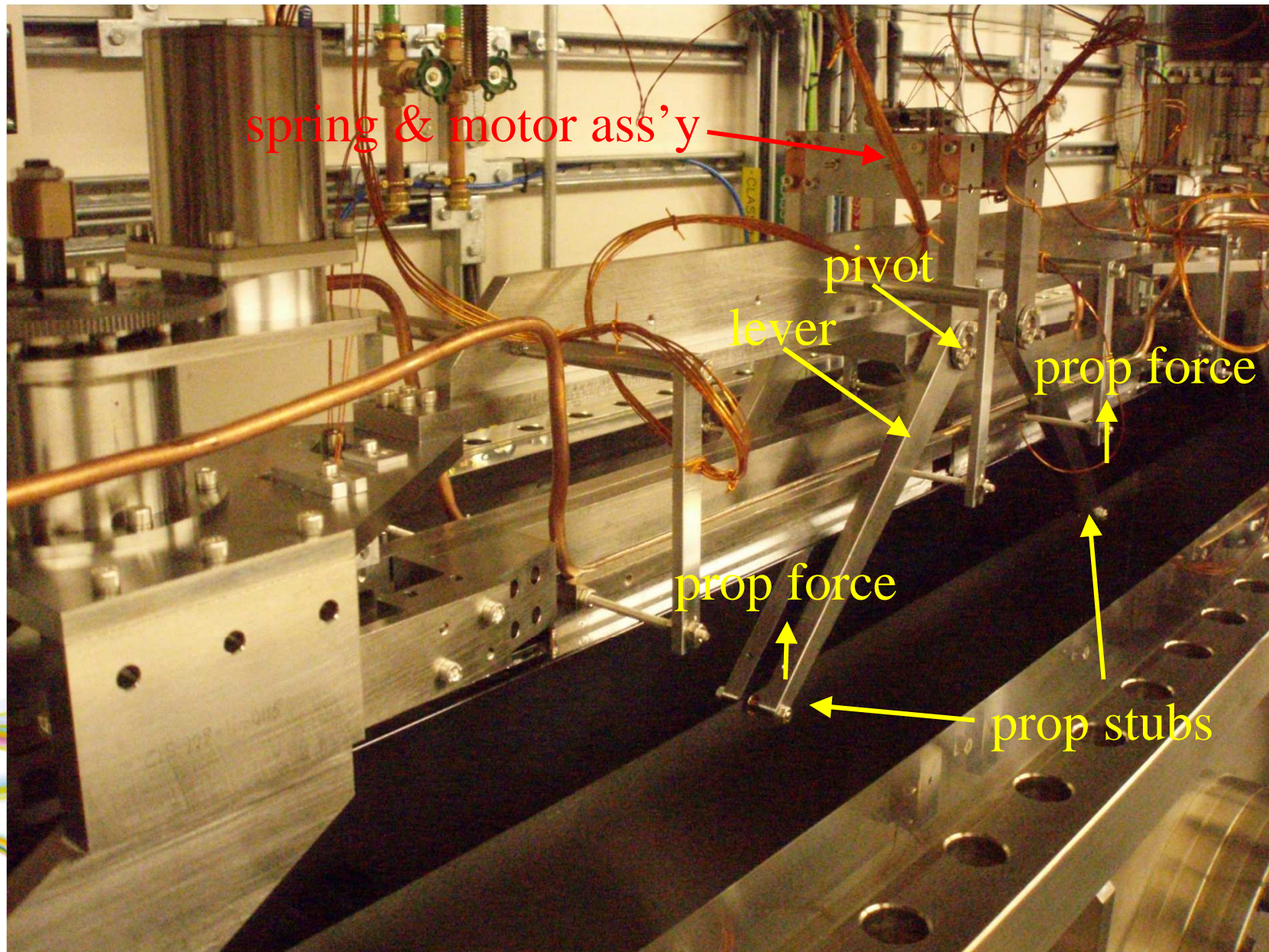


# Diamond beamline I15: Extreme Conditions





## I15 Vertical Focusing Mirror



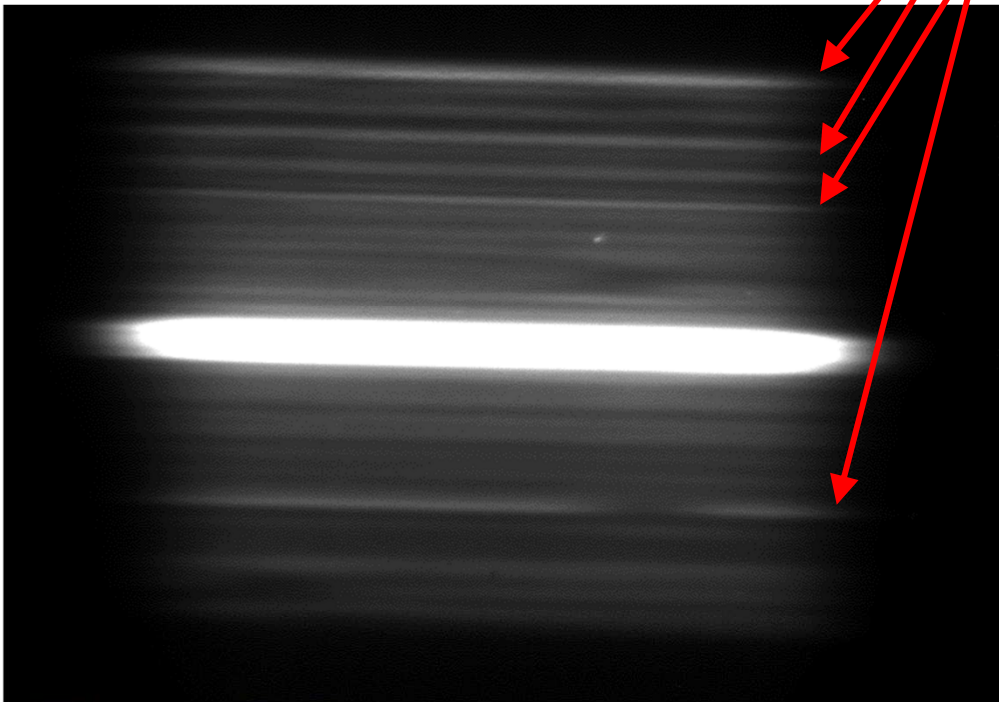


# In-situ optimisation of Extreme Conditions vertical focusing mirror

Initial focusing performance was poor.

Only the central section produced well-focused beam.

beamlets



Focus: whole mirror is illuminated

Focus: only central section is illuminated

Response functions of two bending actuators and the one sag actuator were measured using pencil-beam scans.

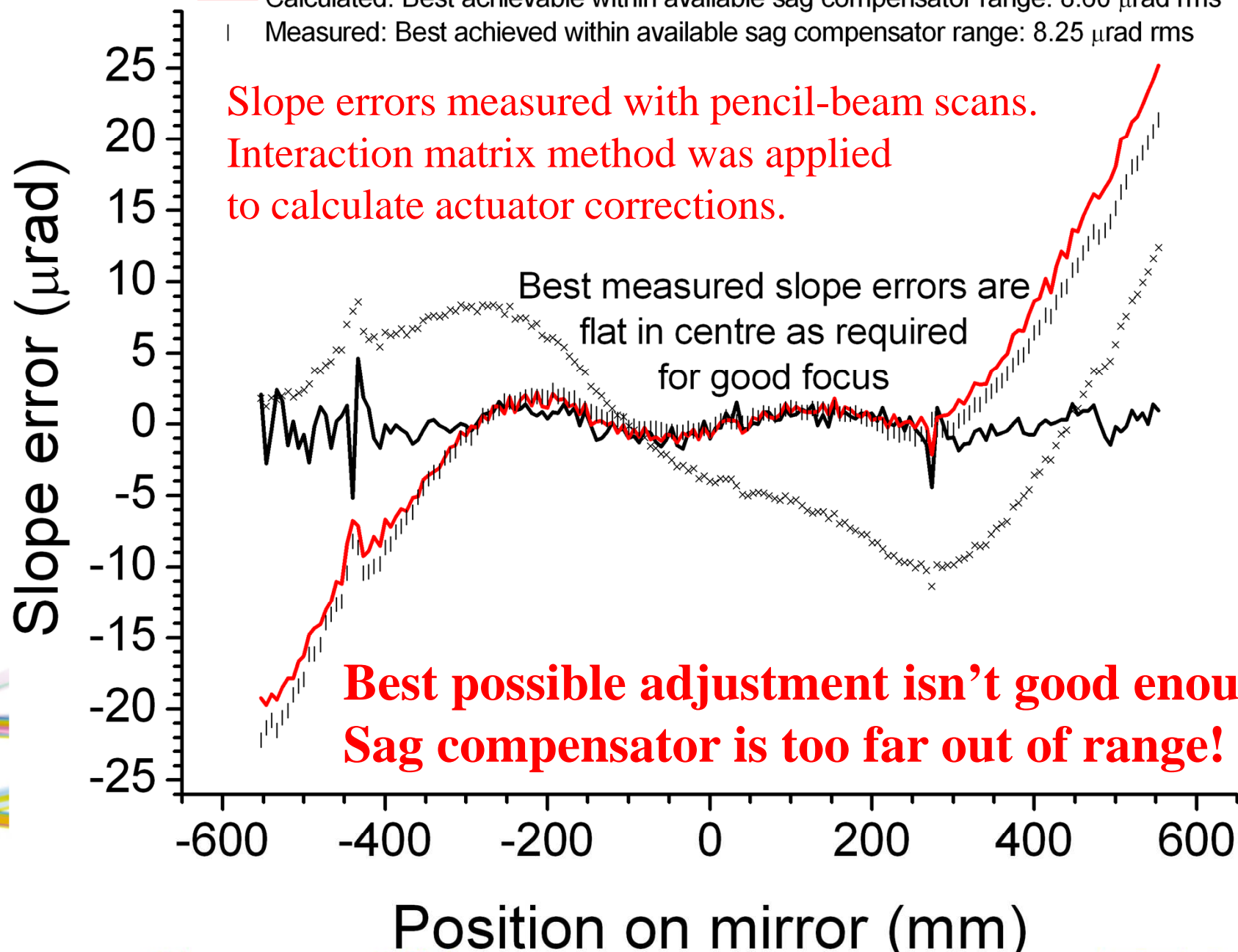
× Measured: initial pencil-beam scan: 6.34  $\mu\text{rad}$  rms

— Calculated: Theoretical best, but sag compensator range too small to reach it: 1.12  $\mu\text{rad}$  rms

— Calculated: Best achievable within available sag compensator range: 8.60  $\mu\text{rad}$  rms

| Measured: Best achieved within available sag compensator range: 8.25  $\mu\text{rad}$  rms

**Slope errors measured with pencil-beam scans.  
Interaction matrix method was applied  
to calculate actuator corrections.**



Without increasing the sag compensator's range,  
simulation showed no way to reduce the slope error on the edges  
by using the benders alone!

→ Preload had to be changed.

Softer preloading springs were placed in the compensator.

→ Compensator range had to be increased.

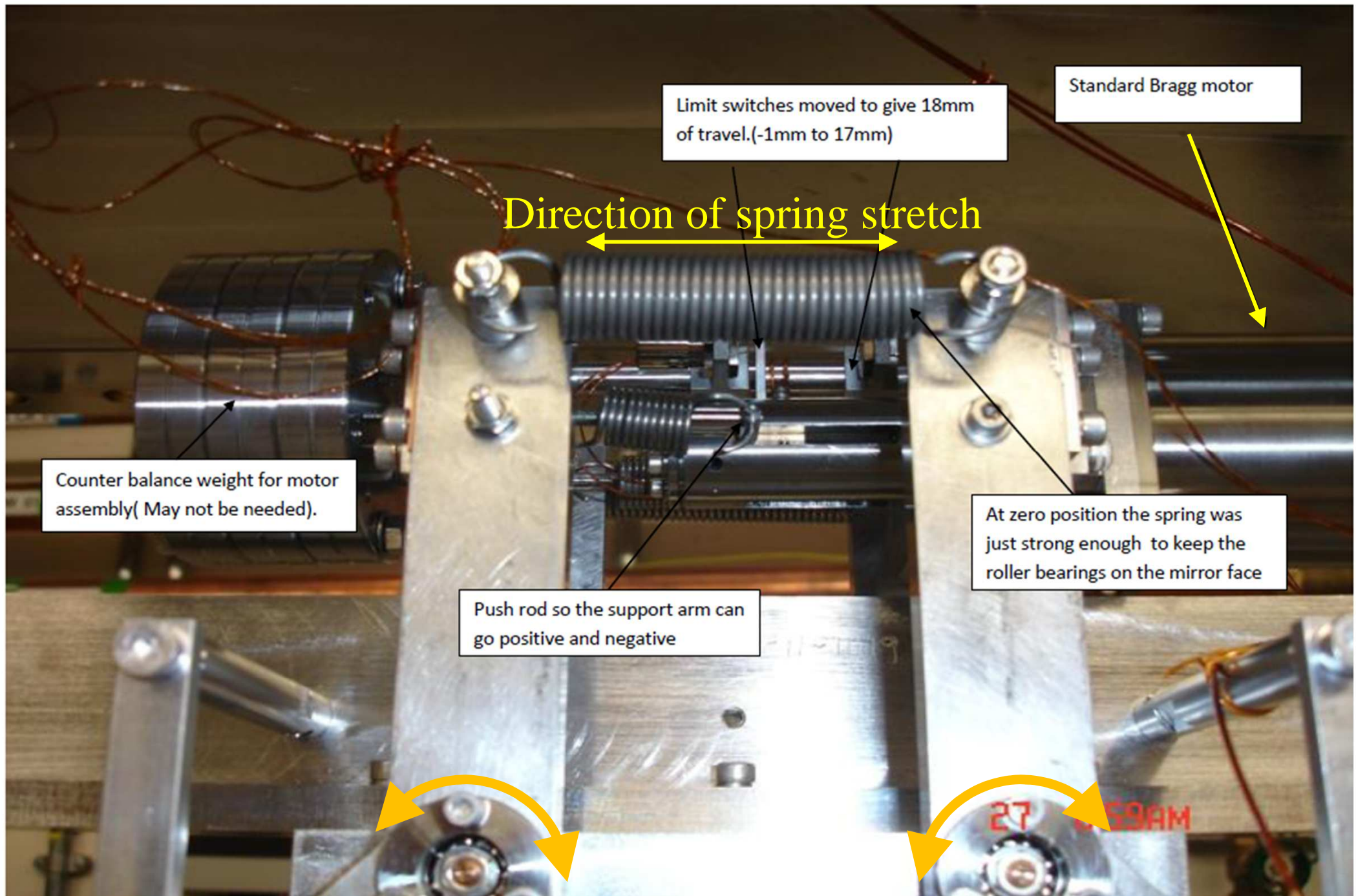
A stiffer spring was placed on the sag compensation motor.

**Result: dramatic improvement!**

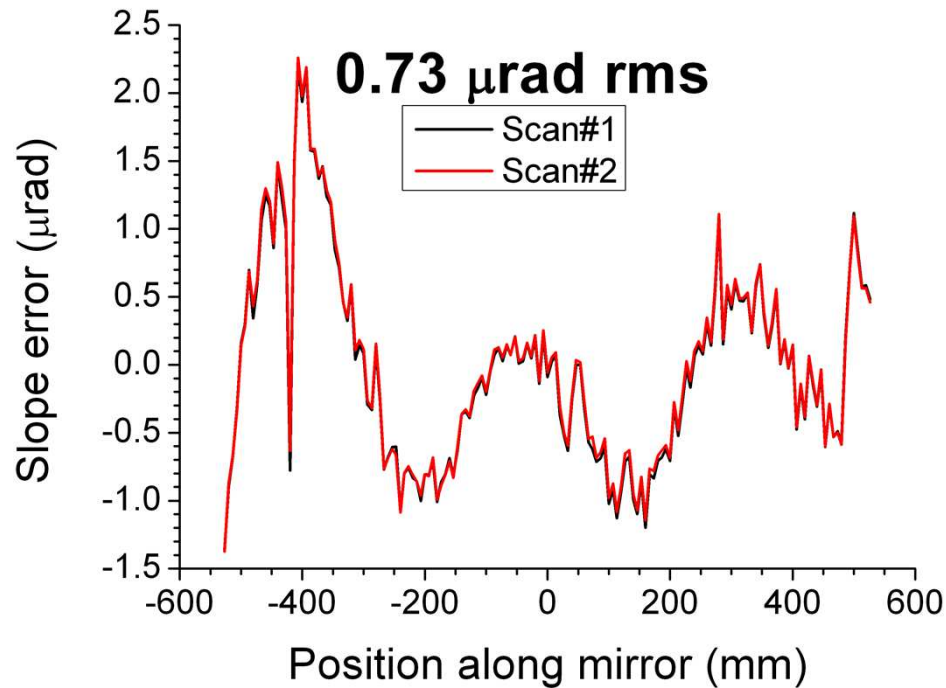




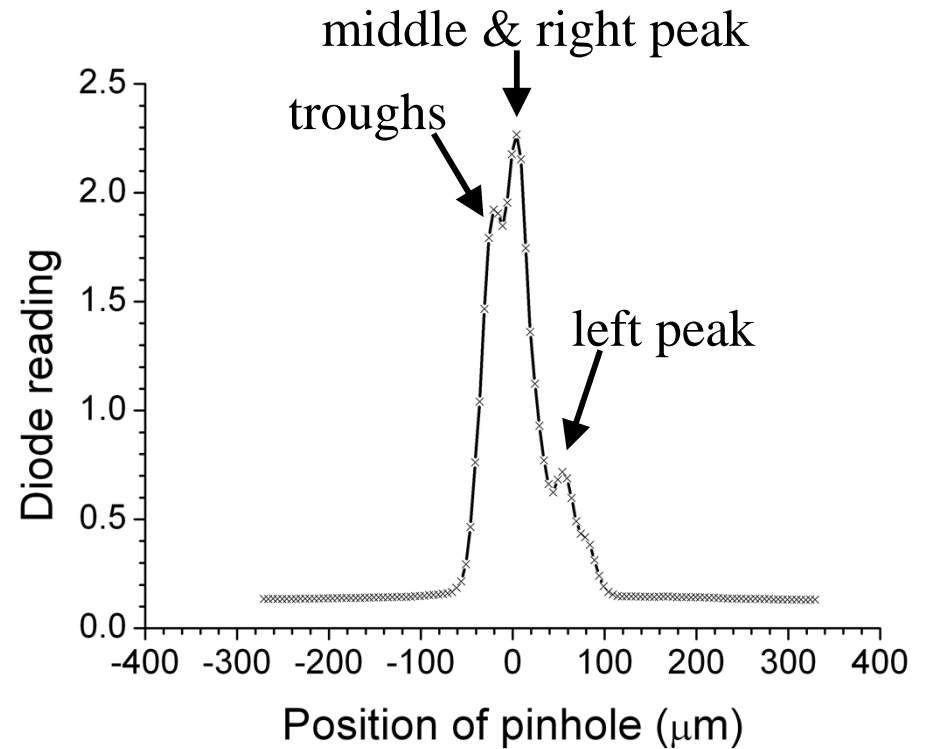
# I15 VFM – New Motor and Spring Assembly



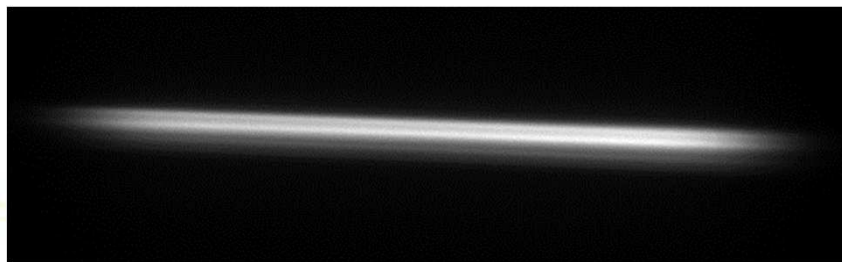
After adjustments to sag compensator springs,



Measured slope error



Vertical beam profile scan with 20  $\mu\text{m}$  pinhole

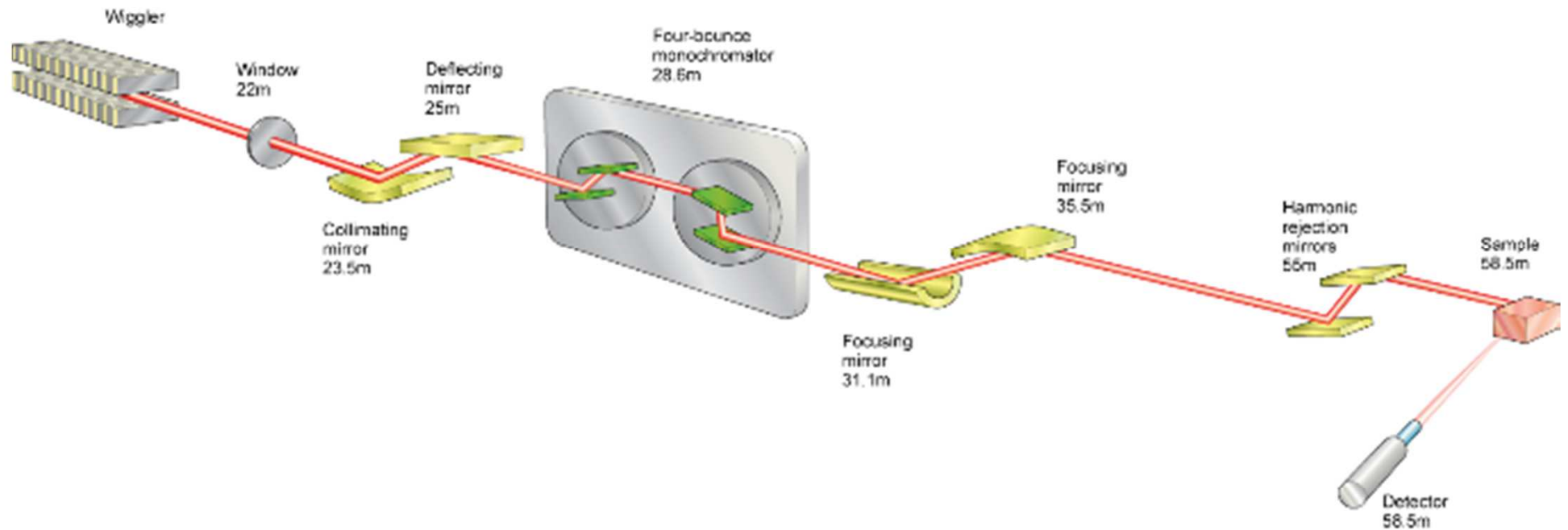


X-ray camera image of focused beam

**Entire mirror is now illuminated.**



# Diamond beamline I20: X-ray Absorption Spectroscopy



# In-situ optimisation of I20 vertical collimating mirror

The theoretical function used to fit the measured mirror slope =  
 $S(x) = \tau + B(x) + G(x)$ :

$S(x)$  is based on the Euler-Bernoulli model of beam bending.

$x$  = position along mirror:  $-L/2 \leq x \leq +L/2$  (see  $L$  below)

$\tau$  = small uniform tilt, used as extra parameter

(Note: good fits obtained with  $\tau$  from 0.59-3.14  $\mu\text{rad}$ )

$B(x)$  = contribution to slope caused by benders (**quadratic**)

$G(x)$  = contribution to slope caused by gravitational sag.

## Properties of mirror:

$L$  = length of mirror = 1390 mm

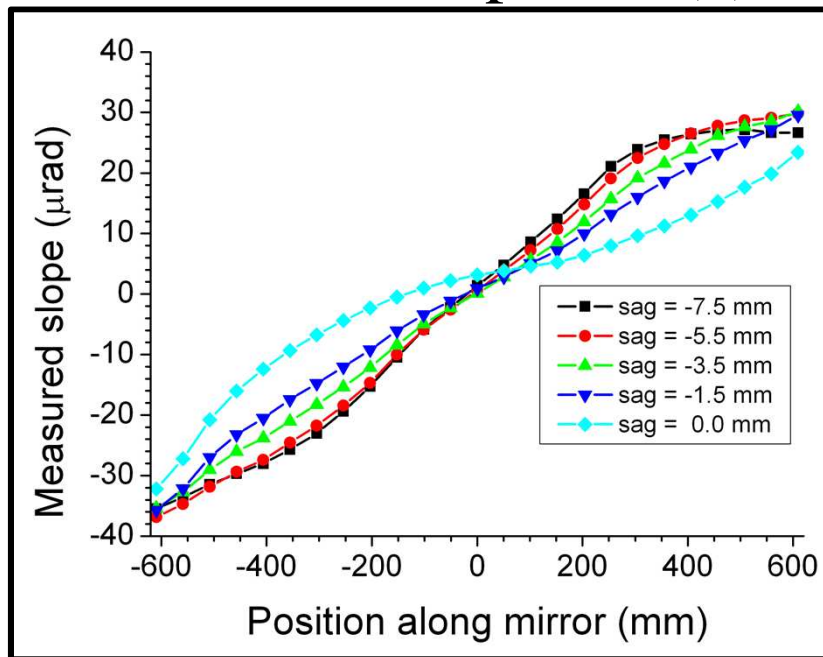
$Y$  = Young's modulus of mirror material =  $1.8 \times 10^{11} \text{ N/m}^2$

$I$  = moment of inertia (ignoring cooling channel grooves)

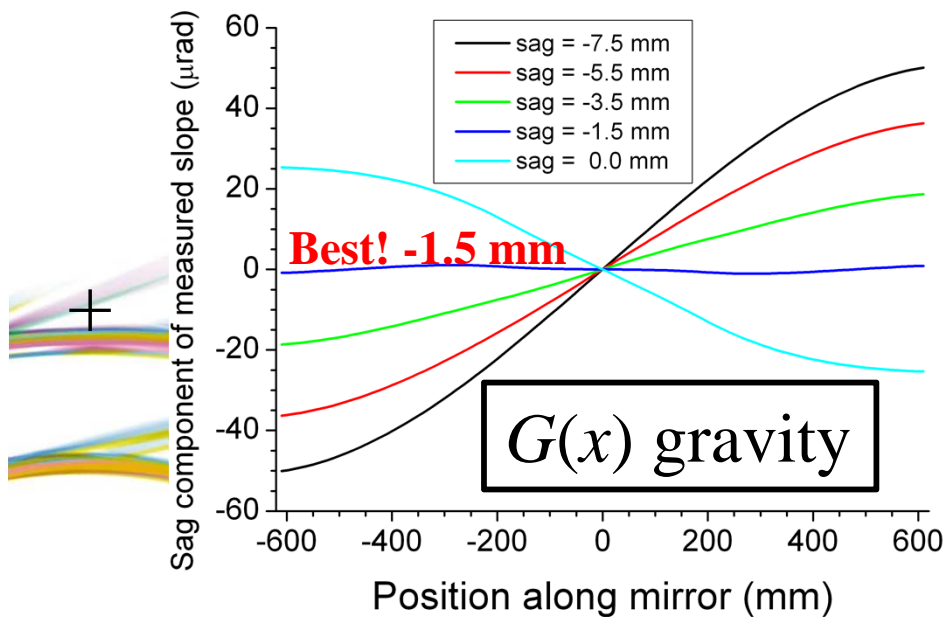
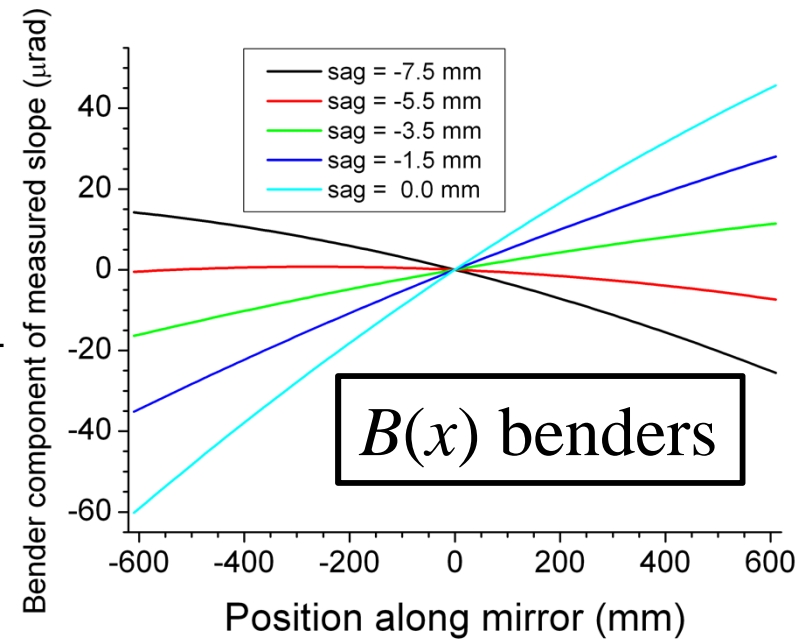
$$\approx 1.26 \times 10^{-6} \text{ m}^4$$

$W$  = weight of mirror  $\approx 133.31 \text{ N}$

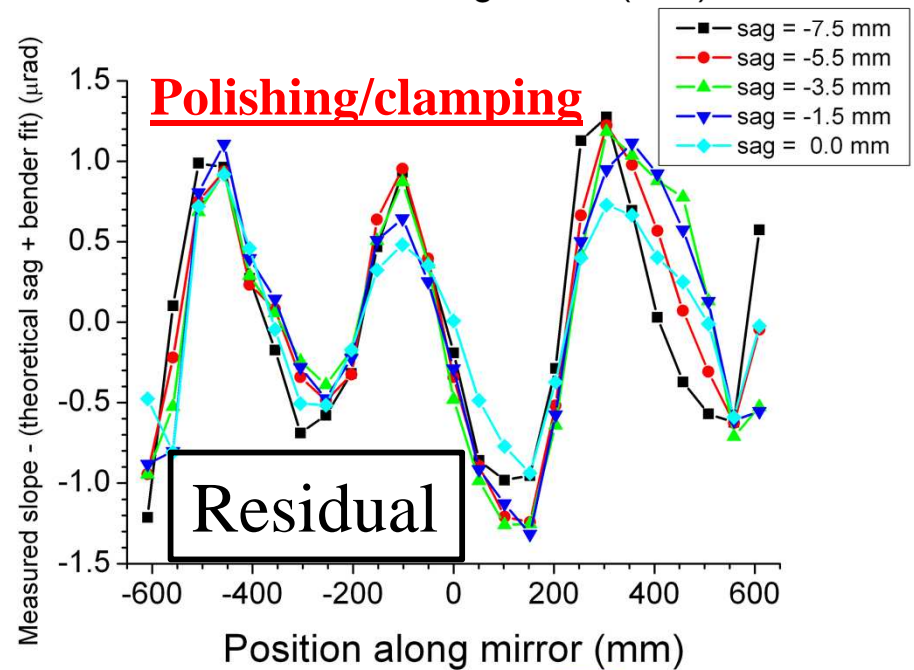
# Fits of measured slope to $S(x)$ :



$= \tau +$



$+$



# Conclusions

- In-situ pencil-beam scans can provide
  - quick and accurate test of mirror figure
  - diagnosis of malfunctioning actuators
  - correct actuator settings of focusing *and collimating* mirrors without complex equipment or major disruption to beamline.
- The optimal sag compensation of the mechanical mirrors could not have been determined ex-situ.
- Bending actuators of mechanical mirrors cannot alone compensate the sag – optimal sag compensation must be found first.



# Acknowledgements

## Diamond Light Source

### Controls

Tom Cobb  
Ronaldo Mercado  
Brian Nutter  
James O'Hea  
Matthew Pearson  
Ulrik Pedersen

### Data Acquisition

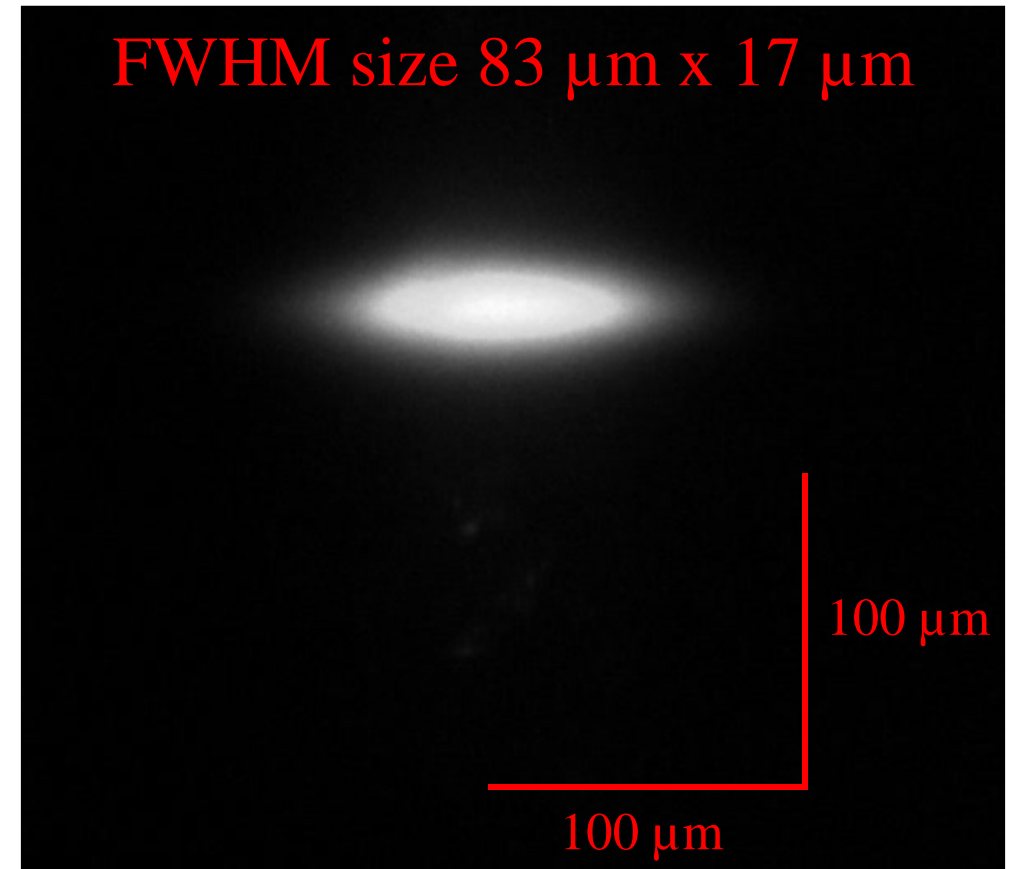
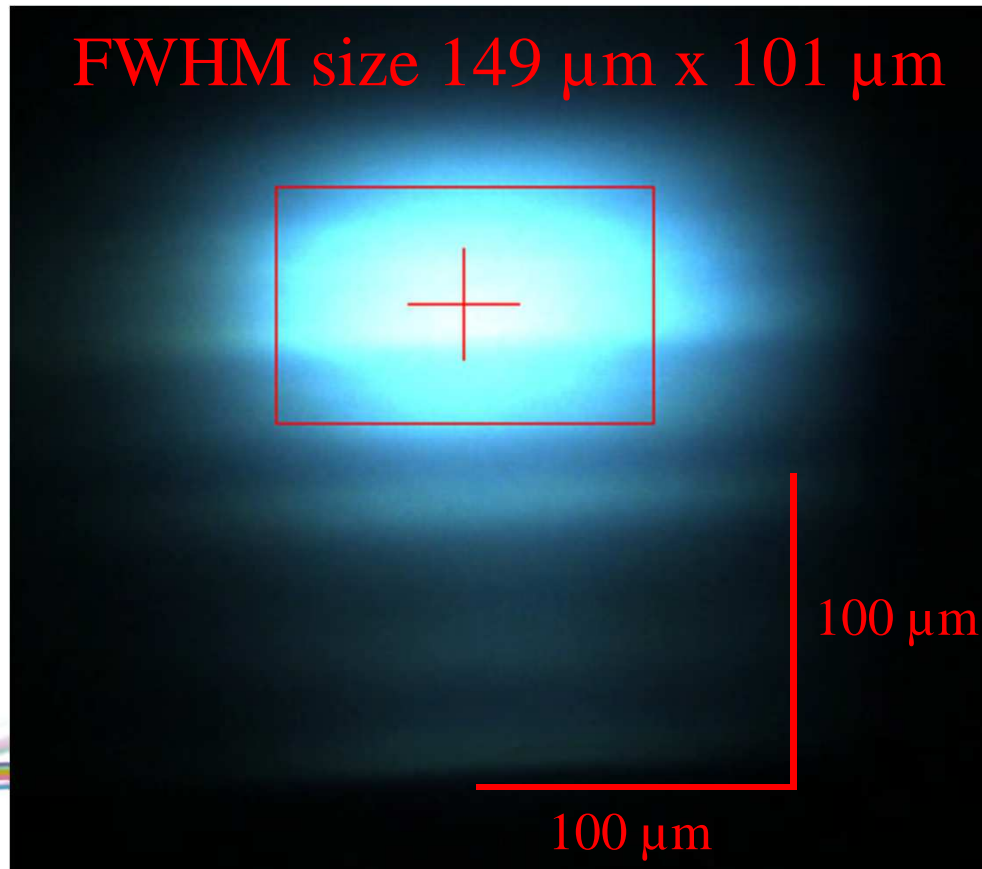
Chris Coles  
Richard Fearn  
Paul Gibbons  
Richard Woolliscroft  
  
I02  
Thomas Sorensen  
Juan Sanchez-Weatherby  
James Sandy  
I03  
Katherine McAuley  
James Nicholson  
Mark Williams

### Diagnostics

Cyrille Thomas  
  
I04  
David Hall  
Ralf Flaig  
  
I15  
Andrew Jephcoat  
Annette Kleppe  
Heribert Wilhelm  
  
I20  
Sofía Díaz-Moreno  
Shusaku Hayama  
Mónica Amboage  
Adam Freeman

Second set of KB bimorph mirrors was taken from I03, repolished and re-installed at I02.

Commissioning late February – early March 2012:



Oct 2011: just before repolishing

Mar 2012: just after repolishing & optimisation with pencil-beam scans

**Present focal size near theoretical limit!**



## Absolute slope error minimisation:

**Sag compensation must be correctly set for good results!**

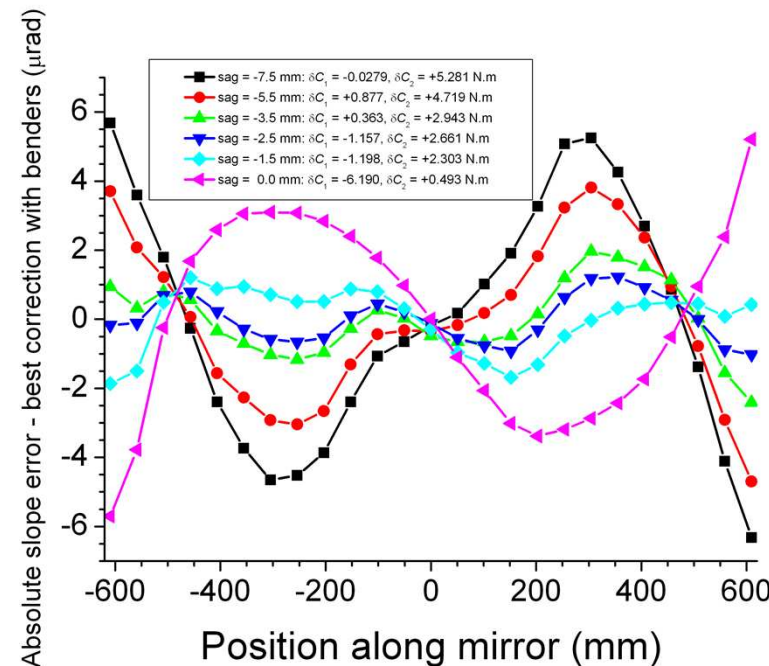
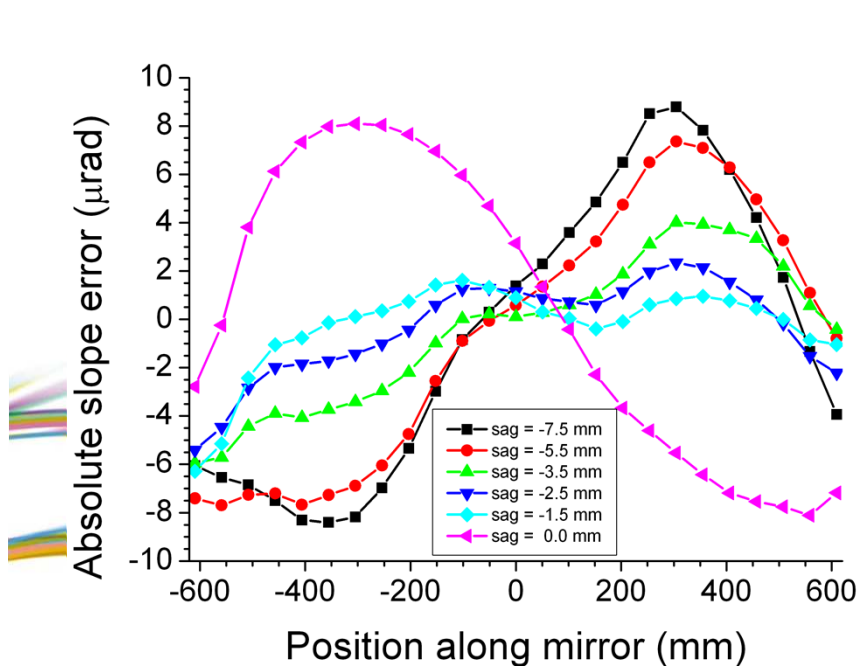
$p = 23355 \text{ mm} = \text{source-mirror distance}$

$\theta = 2.3 \text{ mrad} = \text{angle of incidence}$

→ The ideal slope profile is

$$S_{\text{ideal}} = \{ [p^{1/2} - (p - x \cos \theta)^{1/2}] / [p - x \cos \theta]^{1/2} \} \tan \theta$$

Absolute slope error = measured slope  $S_{\text{meas}}$  – ideal slope  $S_{\text{ideal}}$



sag = -2.5 permits  
best approximation  
to ideal slope when  
benders are used!

Subtraction of best-fit quadratic from  
absolute slope error = best possible  
correction with benders at each sag

Thus, given a single pencil-beam scan of a mirror, we can estimate

- sag under gravity from  $G(x)$
- required corrections to bending forces from  $B(x)$ .

Polishing and clamping errors were not included in this model.

To an extent they can be corrected using the actuators.

**But the residuals of the fits show the polishing and clamping errors that the actuators *cannot* correct.**

